

California State Polytechnic University, Pomona
Aerospace Engineering Department

ARO 4090-01, Spring 2020
Spacecraft Attitude Dynamics and Control

Spacecraft Stability Final Project



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Date: May 15th, 2020

Executive Summary

Methodology

Problem 1: Derive the equations of motion in terms of Eulerian angles and rates, assuming small angles and small rates and $I_x w_0, I_y w_0, \wedge I_z w_0 \ll h_s$.

Equations of Motion: $[I]\dot{w} + \dot{h}_s \hat{h} \circ \wedge + [w]_x ([I]w + h_s \hat{h} \circ \wedge) = T_c + T_d + T_g$

,where T_c is the control torque, T_d is the external disturbance torque, and T_g is the gravity-

gradient torque. The wheel spin-axis is $\hat{h} = \begin{bmatrix} 0 \\ -1 \\ 0 \end{bmatrix}$.

a.) The wheel spin is $h_s \hat{h} = \begin{bmatrix} 0 \\ -h_s \\ 0 \end{bmatrix}$, and the angular rate of the spacecraft in the body frame

with respect to the inertial frame is $w = w_{bl} = \begin{bmatrix} \dot{\phi} - \psi w_0 \\ \dot{\theta} - w_0 \\ \dot{\psi} + \phi w_0 \end{bmatrix}$, derived from $w_{bl} = w_{bo} + C_{bo} w_{ol}$,

where $w_{bo} = \begin{bmatrix} p \\ q \\ r \end{bmatrix} \approx \begin{bmatrix} \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{bmatrix}$ is the angular rate of the spacecraft in the body frame with respect

to the orbital frame, w_0 is the angular rate of the spacecraft in the orbital frame with

respect to the inertial frame, and $C_{bo} = \begin{bmatrix} 1 & \psi & -\theta \\ -\psi & 1 & \phi \\ \theta & -\phi & 1 \end{bmatrix}$ is the rotation matrix from the

body frame to the orbital frame. Therefore, $\dot{w} = \begin{bmatrix} \ddot{\phi} - \dot{\psi} w_0 \\ \ddot{\theta} \\ \ddot{\psi} + \dot{\phi} w_0 \end{bmatrix}$, and the Left-Hand-Side (LHS)

of the equation consists of: $[I]\dot{w} = \begin{bmatrix} I_x(\ddot{\phi} - \dot{\psi} w_0) \\ I_y \ddot{\theta} \\ I_z(\ddot{\psi} + \dot{\phi} w_0) \end{bmatrix}$, $\dot{h}_s \hat{h} = \begin{bmatrix} 0 \\ -\dot{h}_s \\ 0 \end{bmatrix}$,

$[w]_x = \begin{bmatrix} 0 & -(\dot{\psi} + \phi w_0) & \dot{\theta} - w_0 \\ \dot{\psi} + \phi w_0 & 0 & -(\dot{\phi} - \psi w_0) \\ -(\dot{\theta} - w_0) & \dot{\phi} - \psi w_0 & 0 \end{bmatrix}$, $[w]_x [I]w = \begin{bmatrix} (I_y - I_z)(\phi w_0^2 + \dot{\psi} w_0) \\ 0 \\ (I_y - I_x)(\psi w_0^2 - \dot{\phi} w_0) \end{bmatrix}$, and

$[w]_x h_s \hat{h} = \begin{bmatrix} h_s(\dot{\psi} + \phi w_0) \\ 0 \\ h_s(\psi w_0 - \dot{\phi}) \end{bmatrix}$. The RHS knowns are the maximum disturbance T_d and

$$T_g = \frac{3\mu}{R^5} [R_b]_x [I] R_b, \text{ where } [R_b] = \begin{bmatrix} R\theta \\ -R\phi \\ -R \end{bmatrix}, \text{ so } T_g = 3w_0^2 \begin{bmatrix} (I_z - I_y)\phi \\ (I_z - I_x)\theta \\ 0 \end{bmatrix}, \text{ deriving the following}$$

equation of motion:

$$\begin{bmatrix} I_x (\ddot{\phi} - \dot{\psi} w_0) + (I_y - I_z) (\phi w_0^2 + \dot{\psi} w_0) + h_s (\dot{\psi} + \phi w_0) \\ I_y \ddot{\theta} - \dot{h}_s \\ I_z (\ddot{\psi} + \dot{\phi} w_0) + (I_y - I_x) (\psi w_0^2 - \dot{\phi} w_0) + h_s (\psi w_0 - \dot{\phi}) \end{bmatrix} = \begin{bmatrix} T_{cx} + T_{dx} + 3w_0^2 (I_z - I_y) \phi \\ T_{cy} + T_{dy} + 3w_0^2 (I_z - I_x) \theta \\ T_{cz} + T_{dz} \end{bmatrix}$$

b.) Simplifying the previous equation, including \dot{h}_s in T_{cy} and moving the gravity gradient terms onto the left side of the equations, where $k_g = 3(I_x - I_z)w_0^2$:

$$\begin{bmatrix} I_x \ddot{\phi} - (I_x - I_y + I_z) \dot{\psi} w_0 + 4w_0^2 (I_y - I_z) \phi + h_s (\dot{\psi} + \phi w_0) \\ I_y \ddot{\theta} + k_g \theta \\ I_z \ddot{\psi} + (I_x - I_y + I_z) \dot{\phi} w_0 + w_0^2 (I_y - I_x) \psi + h_s (\psi w_0 - \dot{\phi}) \end{bmatrix} = \begin{bmatrix} T_{cx} + T_{dx} \\ T_{cy} + T_{dy} \\ T_{cz} + T_{dz} \end{bmatrix} \quad (1)$$

assuming $I_x w_0, I_y w_0, \wedge I_z w_0 \ll h_s$ our final equations of motion are:

$$\begin{bmatrix} I_x \ddot{\phi} + h_s (\dot{\psi} + \phi w_0) \\ I_y \ddot{\theta} + k_g \theta \\ I_z \ddot{\psi} + h_s (\psi w_0 - \dot{\phi}) \end{bmatrix} = \begin{bmatrix} T_{cx} + T_{dx} \\ T_{cy} + T_{dy} \\ T_{cz} + T_{dz} \end{bmatrix} \quad (2)$$

- c.) A Control System using equation (1) was made using Simulink, whose plots were compared with equation (2), which produced plots with unnoticeable difference and therefore aren't shown.

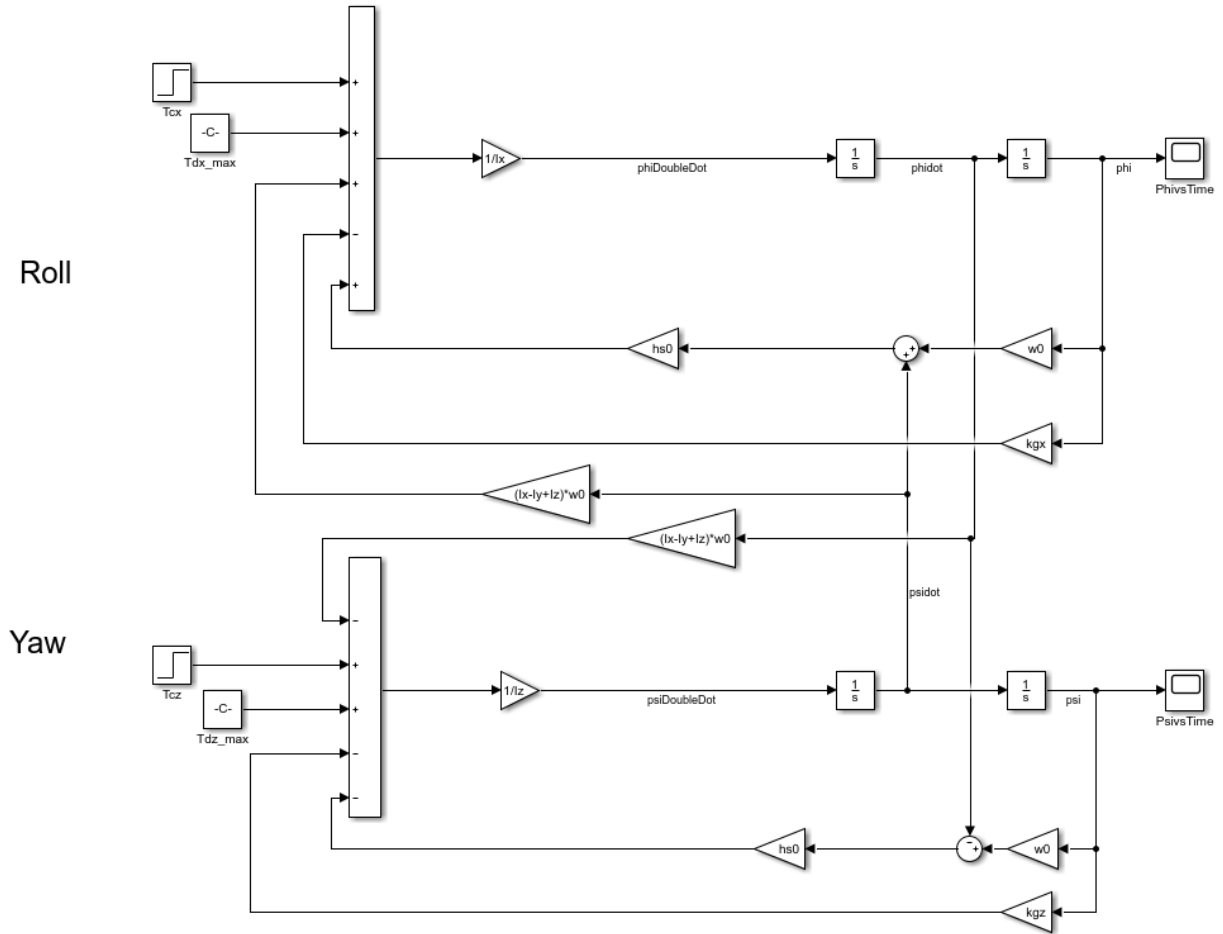


Figure 1.0-1: Coupled Roll/Yaw Simulink Model

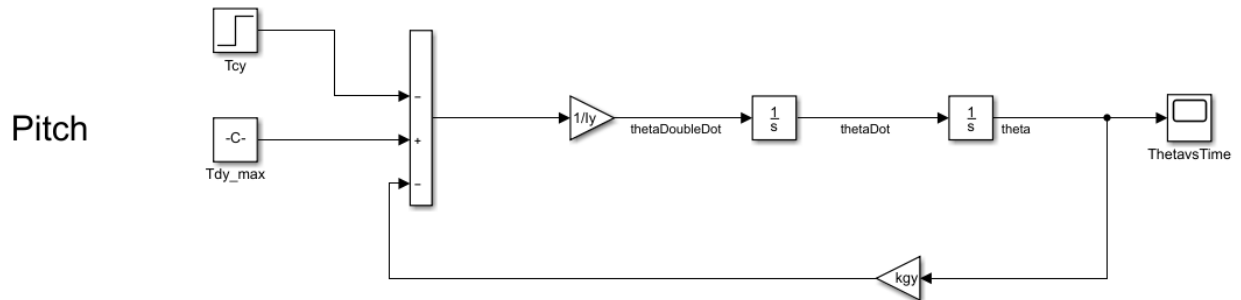


Figure 1.0-2: Uncoupled Pitch Simulink Model

d.) Results from raw Simulink model with 0 control input:

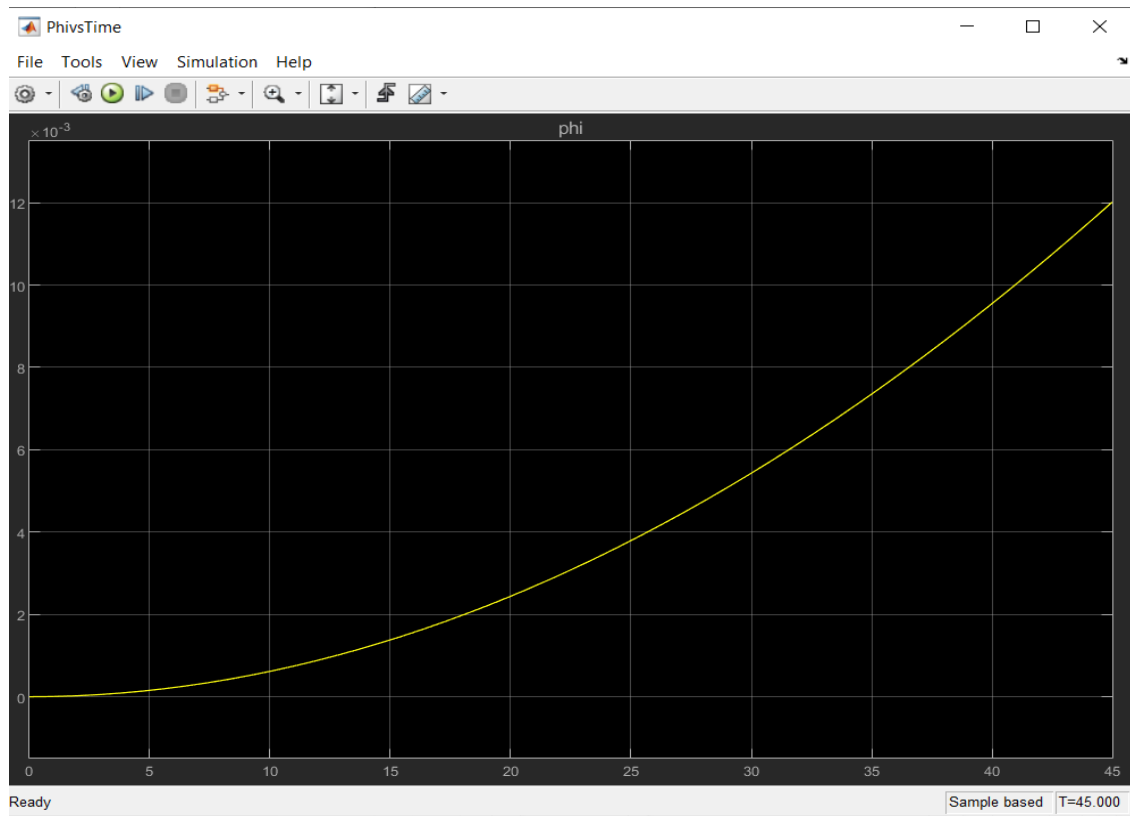


Figure 1.0-3: Roll ϕ (radians) vs Time

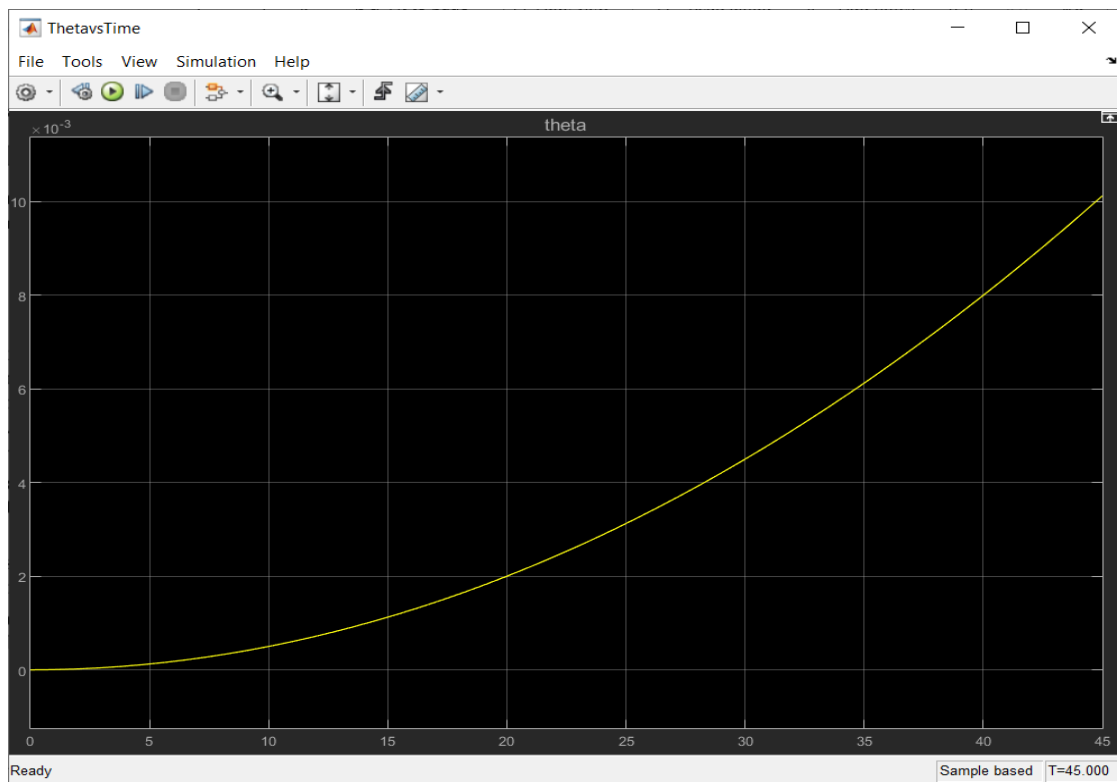


Figure 1.0-4: Pitch θ (radians) vs Time

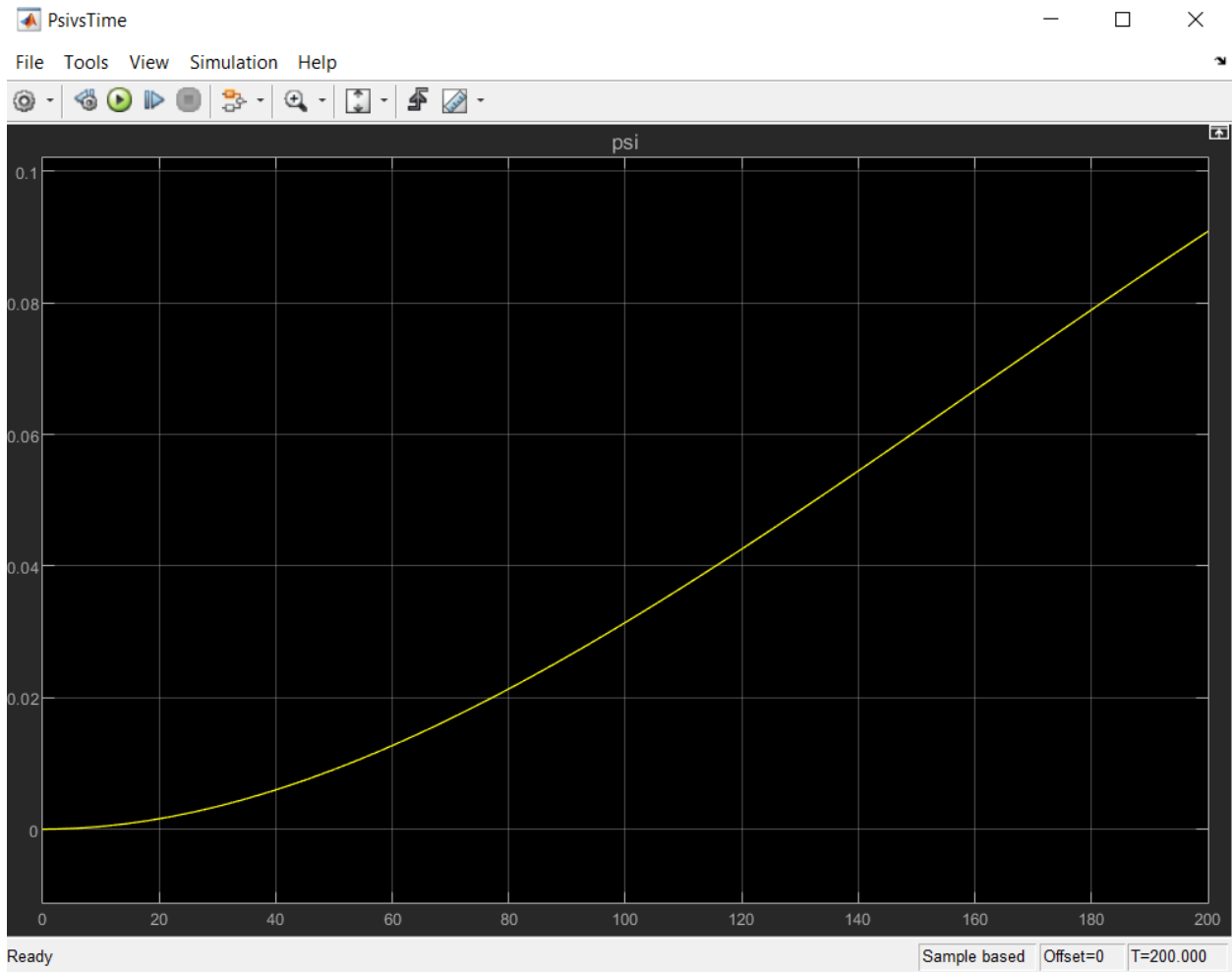


Figure 1.0-5: Yaw ψ (radians) vs Time

Discussion: Derived from the equations of motion, a coupled roll/yaw and an uncoupled pitch Simulink model was created, as show in Figure 1.0-1 and Figure 1.0-2. The results of these control models with 0 control input yields Figures 1.0-3 to 1.0-5, which depict the natural instability of the spacecraft given a maximum disturbance of $5 \mu N \cdot m$. Roll, Pitch and Yaw angles exponentially increase, causing roll to exceed the maximum allowable steady-state error in under 20 seconds, pitch in just under 20 seconds, and yaw in under 3 minutes.

Problem 2: The pitch controller is to be of the modified PD type

$$T_{cy}(t) = k_p^p(\theta_d - \theta) - k_d^p \dot{\theta} \quad (3)$$

- a.) The plant transfer function for the pitch angle is $TF = \frac{\theta(s)}{T_{cy}(s) + T_{dy}(s)} = \frac{1}{I_y s^2 + kg}$ where
This is the form that was used in the Simulink model as seen in Figure 2.0-1 below.

- b.) Implementing the plant transfer function and including the pitch controller and disturbance, we get:

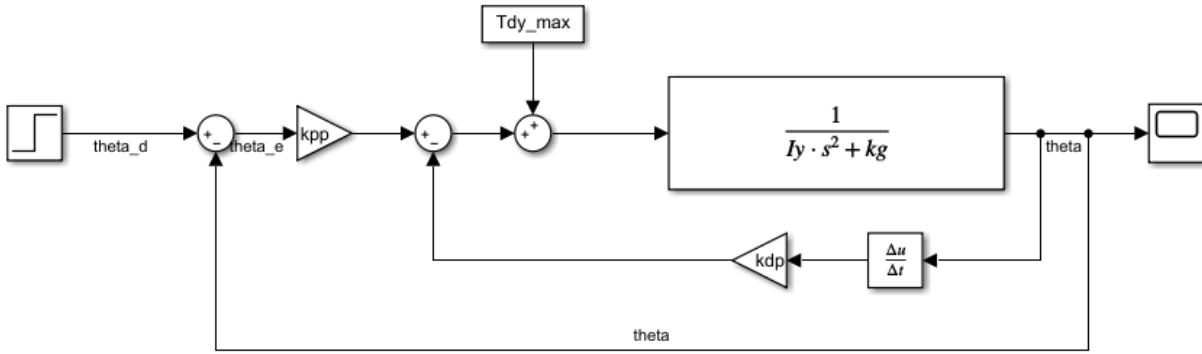


Figure 2.0-1: Controller-Integrated Pitch Angle Simulink Model

- c.) The expression for the steady-state pitch error due to a pitch disturbance torque,

$$T_{dy}(s) = \frac{T_{dy,max}}{s}, \text{ is } \theta_{ss} = \lim_{s \rightarrow 0} sE(s), \text{ where } E(s) = \theta_E(s) = \frac{T_{dy}(s) + T_{cy}(s) + k_d^p s \theta(s)}{k_p^p} = \frac{\frac{T_{dy,max}}{s} + T_{cy}(s) + k_d^p s \theta(s)}{k_p^p}$$

is derived from equation (3).

Therefore $\theta_{ss} = \lim_{s \rightarrow 0} sE(s) = \frac{T_{dy,max}}{k_p^p} = \frac{5 \cdot 10^{-6}}{0.0015} = 0.00333$. Alternatively, we can assume the steady-state error is 0 because we have a type 2 system and step input.

- d.) The block diagram in Figure 2.0-1 was reduced by moving the summing-point of the disturbance torque before the summing-point of the derivative gain, and then obtaining the transfer function for the inner closed-loop, yielding:

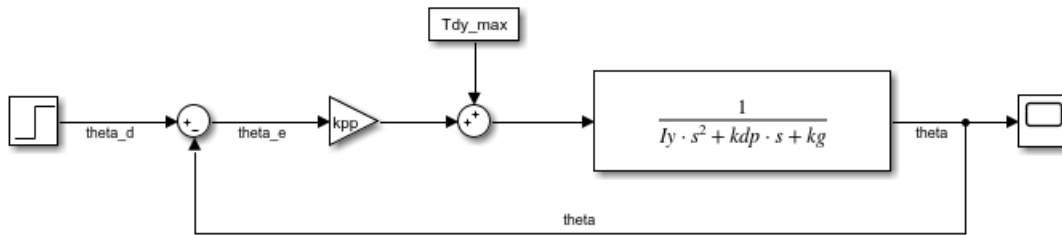


Figure 2.0-2: Reduced Pitch Angle Block Diagram

Therefore the closed-loop transfer function from desired pitch angle to actual, neglecting

disturbance, is $\frac{\theta(s)}{\theta_d(s)} = \frac{k_p^p}{I_y s^2 + k_d^p s + kg + k_p^p}$

e.) The pitch control gains are $k_p^p = w_n^2 I_y$, $k_d^p = 2\zeta w_n I_y$, where ζ is solved using closed-loop pitch specification $M_p = \exp\left(\frac{-\pi\zeta}{\sqrt{1-\zeta^2}}\right) = 0.30$, and w_n is solved using

$$t_s = \frac{\ln(0.02\sqrt{1-\zeta^2})}{-\zeta w_n} = 200. \text{ Yielding } k_p^p = 0.00155 \text{ and } k_d^p = 0.00199.$$

Discussion: The pitch angle is successfully controlled to its desired input with a settling time of 200 seconds and maximum overshoot of 30%. In Figure 2.0-3 below is a step response to a desired theta value of 0.5 radians.

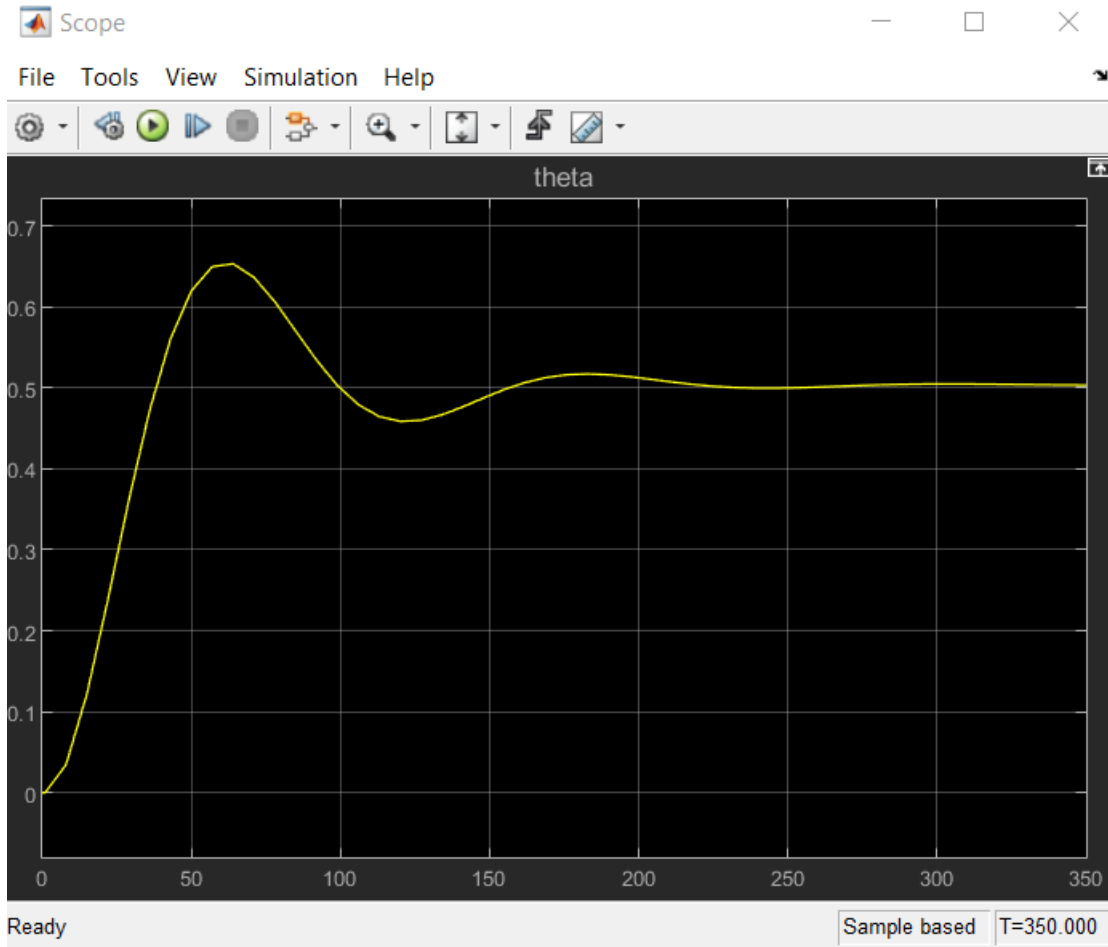


Figure 2.0-3: Pitch Angle Step Response

Problem 3: The control law for the roll/yaw loop are the following

$$T_{cx} = -(k_p^{ry} \phi + k_d^{ry} \dot{\phi}) \quad (4)$$

$$T_{cz} = \alpha (k_p^{ry} \phi + k_d^{ry} \dot{\phi}) \quad (5)$$

where $\alpha > 0$, $k_p^{ry} > 0 \wedge k_d^{ry} > 0$.

a.)

b.) Substituting equation (4) and (5) into equation (2) and taking the Laplace transform gives us the closed-loop roll/yaw equations of motion in Laplace domain in the matrix form:

$$\begin{pmatrix} \phi(s) \\ \psi(s) \end{pmatrix} = \frac{1}{\Delta} \dot{\mathbf{i}},$$

Where

$$\Delta = s^4$$

$$+ s^3 k_d^{ry}$$

$$+ s^2 \dot{\mathbf{i}}$$

$$+ s^1 \dot{\mathbf{i}}$$

$$+ s^0 \dot{\mathbf{i}}$$

c.) Therefore for $\alpha > 0, k_p^{ry} > 0 \wedge k_p^{ry} > 0$ the transfer functions for roll and yaw would be:

$$\frac{\phi(s)}{T_{dx}(s)} = \frac{I_x}{\Delta} \left(s^2 + \frac{h_s w_0}{I_z} \right)$$

$$\frac{\phi(s)}{T_{dz}(s)} = \frac{-I_z}{\Delta} \left(\frac{s h_s}{I_x} \right)$$

$$\frac{\psi(s)}{T_{dx}(s)} = \frac{I_x}{\Delta} \left(\frac{s}{I_z} (h_s + \alpha k_d^{ry}) + \alpha k_p^{ry} \right)$$

$$\frac{\phi(s)}{T_{dz}(s)} = \frac{-I_z}{\Delta} \left(s^2 + \frac{1}{I_x} (h_s w_0 + k_p^{ry} + s * k_d^{ry}) \right)$$

d.)